



US009359540B2

(12) **United States Patent**
Rached(10) **Patent No.:** **US 9,359,540 B2**
(45) **Date of Patent:** ***Jun. 7, 2016**(54) **TERNARY HEAT-TRANSFER FLUIDS
COMPRISING DIFLUOROMETHANE,
PENTAFLUOROETHANE AND
TETRAFLUOROPROPENE**(71) Applicant: **Arkema France**, Colombes (FR)(72) Inventor: **Wissam Rached**, Chaponost (FR)(73) Assignee: **ARKEMA FRANCE**, Colombes (FR)(*) Notice: Subject to any disclaimer, the term of this
patent is extended or adjusted under 35
U.S.C. 154(b) by 0 days.This patent is subject to a terminal dis-
claimer.(21) Appl. No.: **14/645,066**(22) Filed: **Mar. 11, 2015**(65) **Prior Publication Data**

US 2015/0184052 A1 Jul. 2, 2015

Related U.S. Application Data(63) Continuation of application No. 13/696,870, filed as
application No. PCT/FR2011/050882 on Apr. 18,
2011, now Pat. No. 9,057,010.(30) **Foreign Application Priority Data**

May 11, 2010 (FR) 10 53671

(51) **Int. Cl.****C09K 5/04** (2006.01)**F25B 45/00** (2006.01)(52) **U.S. Cl.**CPC **C09K 5/045** (2013.01); **F25B 45/00**
(2013.01); **C09K 2205/126** (2013.01); **C09K**
2205/22 (2013.01); **C09K 2205/40** (2013.01)(58) **Field of Classification Search**CPC C09K 5/045; C09K 2205/126; C09K
2205/122; F25B 1/00
USPC 252/67, 68; 62/467, 468
See application file for complete search history.(56) **References Cited****U.S. PATENT DOCUMENTS**2,438,120 A 3/1948 Freygang
2,834,748 A 5/1958 Bailey et al.
2,846,458 A 8/1958 Haluska et al.
2,917,480 A 12/1959 Bailey et al.
5,399,631 A 3/1995 Egawa et al.
5,497,631 A 3/1996 Lorentzen et al.
5,643,492 A 7/1997 Shiflett
5,688,432 A 11/1997 Pearson
5,722,256 A 3/1998 Shiflett
5,744,052 A 4/1998 Bivens
6,454,960 B1 9/2002 Sunaga et al.
6,508,950 B1 1/2003 Lim et al.
6,589,355 B1 7/2003 Thomas et al.
6,655,160 B2 12/2003 Roberts
7,914,696 B2 3/2011 Low et al.
8,070,977 B2 12/2011 Rached8,075,798 B2 12/2011 Rached
8,142,680 B2 3/2012 Rached
8,246,850 B2 8/2012 Rached
8,443,624 B2* 5/2013 Yamashita et al. 62/498
8,496,845 B2 7/2013 Tsuchiya et al.
9,057,010 B2 6/2015 Rached
2006/0243944 A1 11/2006 Minor et al.
2006/0269484 A1 11/2006 Knopeck et al.
2007/0108403 A1 5/2007 Sievert et al.
2008/0230738 A1 9/2008 Minor et al.
2009/0158771 A1 6/2009 Low et al.
2009/0249864 A1 10/2009 Minor et al.
2009/0250650 A1 10/2009 Minor et al.
2009/0278072 A1 11/2009 Minor
2009/0305876 A1 12/2009 Singh et al.
2010/0044619 A1 2/2010 Hulse et al.
2010/0044620 A1 2/2010 Rached
2010/0122545 A1 5/2010 Minor et al.
2011/0079042 A1 4/2011 Yamashita et al.
2011/0095224 A1 4/2011 Rached
2011/0108756 A1 5/2011 Tsuchiya et al.
2011/0162410 A1 7/2011 Low
2011/0173997 A1 7/2011 Low et al.
2011/0186772 A1 8/2011 Rached
2011/0219792 A1 9/2011 Rached
2011/0219815 A1 9/2011 Yana Motta et al.
2011/0240254 A1 10/2011 Rached
2011/0284181 A1 11/2011 Rached
2012/0049104 A1 3/2012 Rached
2012/0056123 A1 3/2012 Rached
2012/0097885 A9 4/2012 Hulse et al.
2012/0144857 A1 6/2012 Rached

(Continued)

FOREIGN PATENT DOCUMENTSEP 0 509 673 A1 10/1992
EP 0 811 670 A1 12/1997

(Continued)

OTHER PUBLICATIONS

English Translation of WO 2009/154149, Dec. 23, 2009.*

(Continued)

Primary Examiner — Douglas McGinty(74) *Attorney, Agent, or Firm* — Buchanan Ingersoll &
Rooney P.C.(57) **ABSTRACT**A ternary composition including: from 5 to 50% of difluo-
romethane; from 2 to 20% of pentafluoroethane; and from 30
to 90% of tetrafluoropropene. The tetrafluoropropene may be
1,3,3,3-tetrafluoropropene or 2,3,3,3-tetrafluoropropene.
This composition can be used as a heat-transfer fluid in vapor
compression circuit. A process for heating or cooling a fluid
or a body using a vapor compression circuit containing a heat
transfer fluid, said process successively including evapora-
tion of the heat transfer fluid, compression of the heat transfer
fluid, condensation of the heat fluid and depressurization of
the heat transfer fluid, in which the heat transfer fluid is a
composition as described.**19 Claims, No Drawings**

(56)

References Cited

U.S. PATENT DOCUMENTS

2012/0151959	A1	6/2012	Rached
2012/0153213	A1	6/2012	Rached
2012/0159982	A1	6/2012	Rached
2012/0161064	A1	6/2012	Rached
2012/0167615	A1	7/2012	Rached
2012/0255316	A1	10/2012	Andre et al.
2012/0312048	A1	12/2012	Poole et al.
2013/0055733	A1	3/2013	Rached
2013/0055738	A1	3/2013	Rached
2013/0055739	A1	3/2013	Rached
2013/0061613	A1	3/2013	Rached
2013/0096218	A1	4/2013	Rached et al.
2013/0145778	A1	6/2013	Yana Motta et al.
2013/0193369	A1	8/2013	Low
2013/0255284	A1	10/2013	Rached
2014/0075969	A1	3/2014	Guerin et al.
2014/0137578	A1	5/2014	Yana Motta et al.
2014/0223927	A1	8/2014	Pottker et al.
2015/0135765	A1*	5/2015	Yana Motta et al. 62/498
2015/0291869	A1	10/2015	Boussand

FOREIGN PATENT DOCUMENTS

KR	2001-0044992	A	6/2001
WO	WO 2004/037913	A2	5/2004
WO	WO 2004/037913	A3	5/2004
WO	WO 2005/105947	A2	11/2005
WO	WO 2005/105947	A3	11/2005
WO	WO 2007/002625	A2	1/2007
WO	WO 2007/053697	A2	5/2007
WO	WO 2007/053697	A3	5/2007
WO	WO 2007/126414	A2	11/2007
WO	WO 2007/126414	A3	11/2007
WO	WO 2009/047542	A1	4/2009
WO	WO 2009/151669	A1	12/2009
WO	WO 2009/154149	A1	12/2009

WO	WO/2010/059677	A2	5/2010
WO	WO 2010/064005	A1	6/2010
WO	WO 2010/002014	A	10/2010
WO	WO 2010/129461	A2	11/2010
WO	WO 2010/129920	A1	11/2010
WO	WO 2011/073934	A1	6/2011
WO	WO 2011/077088	A1	6/2011
WO	WO 2011/107698	A2	9/2011
WO	WO 2011/107698	A3	9/2011
WO	WO 2011/141654	A2	11/2011
WO	WO 2011/141654	A3	11/2011
WO	WO 2011/141656	A2	11/2011
WO	WO 2011/141656	A3	11/2011

OTHER PUBLICATIONS

Third Party Observation mailed on May 26, 2014, by the European Patent Office in corresponding European Patent Application No. 11731420.3. (4 pages).

Takizawa, K., et al., "Flammability Assessment of CH₂=CFCF₃: Comparison with Fluoroalkenes and Fluoroalkanes", Journal of Hazardous Materials, vol. 172, No. 2-3, 18, Aug. 18, 2009, pp. 1329-1338, XP026719989, Elsevier B.V.

"Definitions: Humidity," Healthy Heating, May 18, 2008, 4 pages, XP002594956, http://web.archive.org/web/20080518174151/http://www.healthyheating.com/Thermal_Comfort_Working_Copy/Definitions/humidity.htm.

International Search Report issued in PCT/FR2011/050882 (French and English-language versions), mailed Dec. 28, 2011, 5 pages, European Patent Office, Rijswijk, NL.

Translated Excerpt from official Action issued Mar. 3, 2015 in corresponding Japanese Patent Application No. 2013-0509593, Japan Patent Office, 1 page.

Takizawa, K., et al., "Flammability Assessment of CH₂=CFCF₃ (R-1234yf) and its Mixtures with CH₂F₂ (R-32): 2010 International Symposium on Next-generation Air Conditioning and Refrigeration Technology," Tokyo, JP, Feb. 17-19, 2010, pp. 1-8.

* cited by examiner

1

**TERNARY HEAT-TRANSFER FLUIDS
COMPRISING DIFLUOROMETHANE,
PENTAFLUOROETHANE AND
TETRAFLUOROPROPENE**

**CROSS REFERENCE TO RELATED
APPLICATIONS**

The present application is a continuation of U.S. application Ser. No. 13/696,870 (now U.S. Pat. No. 9,057,010 and issued on Jun. 16, 2015), filed on Nov. 8, 2012, which is U.S. National Phase of International Application No. PCT/FR2011/050882, filed on Apr. 18, 2011 claims the benefit of French Application No. 1053671, filed on May 11, 2010. The entire contents of each of U.S. application Ser. No. 13/696,870, U.S. Pat. No. 9,057,010, International Application No. PCT/FR2011/050882, and French Application No. 1053671 are hereby incorporated herein by reference in their entirety.

FIELD OF THE INVENTION

The present invention relates to transfer fluids based on difluoromethane, pentafluoroethane and tetrafluoropropene, which have high performance qualities and a low GWP, and are thus suitable for replacing the usual coolants.

TECHNICAL BACKGROUND

Fluids based on fluorocarbon compounds are widely used in vapor-compression heat transfer systems, especially air-conditioning, heat-pump, refrigerator or freezer devices. The common feature of these devices is that they are based on a thermodynamic cycle comprising vaporization of the fluid at low pressure (in which the fluid absorbs heat); compression of the vaporized fluid up to a high pressure; condensation of the vaporized fluid to liquid at high pressure (in which the fluid expels heat); and depressurization of the fluid to complete the cycle.

The choice of a heat transfer fluid (which may be a pure compound or a mixture of compounds) is dictated firstly by the thermodynamic properties of the fluid, and secondly by additional constraints. Thus, a particularly important criterion is the impact of the fluid under consideration on the environment. In particular, chlorinated compounds (chlorofluorocarbons and hydrochlorofluorocarbons) have the drawback of damaging the ozone layer. Non-chlorinated compounds such as hydrofluorocarbons, fluoroethers and fluoroolefins are therefore now generally preferred to chlorinated compounds.

Heat transfer fluids that are currently used are HFC-134a, R404a (ternary mixture of 52% HFC-143a, 44% HFC-125 and 4% HFC-134a) and R407c (ternary mixture of 52% HFC-134a, 25% HFC-125 and 23% HFC-32).

It is, however, necessary to develop other heat transfer fluids that have a lower global warming potential (GWP) than that of the above fluids, and which have equivalent and preferably improved performance qualities.

Document US 2009/0 250 650 describes various fluoroolefin-based compositions and their use as heat transfer fluids. In particular, the document describes the mixture consisting of HFC-32, HFC-125 and HFO-1234ze and also the mixture consisting of HFC-32, HFC-125 and HFO-1234yf. The compositions indicated as being preferred are the following:

23% HFC-32, 25% HFC-125 and 52% HFO-1234ze;
30% HFC-32, 50% HFC-125 and 20% HFO-1234ze;
40% HFC-32, 50% HFC-125 and 10% HFO-1234yf;
23% HFC-32, 25% HFC-125 and 52% HFO-1234yf;

2

15% HFC-32, 45% HFC-125 and 40% HFO-1234yf; and
10% HFC-32, 60% HFC-125 and 30% HFO-1234yf.

Document WO 2010/002 014 describes a non-flammable coolant based on HFC-32, HFC-125 and HFO-1234yf. Several compositions are disclosed and especially that comprising 15% HFC-32, 25% HFC-125 and 60% HFO-1234yf.

However, there is still a need to develop other heat transfer fluids that have a relatively low GWP and that have better energy performance qualities than the heat transfer fluids of the prior art.

SUMMARY OF THE INVENTION

The invention relates firstly to a ternary composition comprising:

from 5% to 50% of difluoromethane;
from 2% to 20% of pentafluoroethane; and
from 30% to 90% of tetrafluoropropene.

According to one embodiment, the tetrafluoropropene is 1,3,3,3-tetrafluoropropene.

According to another embodiment, the tetrafluoropropene is 2,3,3,3-tetrafluoropropene.

According to one embodiment, the composition comprises:

from 15% to 35% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 45% to 80% of 2,3,3,3-tetrafluoropropene;
and preferably:
from 18% to 25% of difluoromethane;
from 8% to 20% of pentafluoroethane; and
from 55% to 74% of 2,3,3,3-tetrafluoropropene.

According to one embodiment, the composition comprises:

from 15% to 50% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 30% to 80% of 1,3,3,3-tetrafluoropropene;
and preferably:
from 30% to 40% of difluoromethane;
from 8% to 20% of pentafluoroethane; and
from 40% to 62% of 1,3,3,3-tetrafluoropropene.

According to one embodiment, the composition comprises:

from 5% to 30% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 50% to 90% of 1,3,3,3-tetrafluoropropene;
and preferably:
from 5% to 20% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 60% to 90% of 1,3,3,3-tetrafluoropropene.

According to one embodiment, the composition comprises:

from 20% to 40% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 40% to 75% of 1,3,3,3-tetrafluoropropene;
and preferably:
from 25% to 40% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 40% to 70% of 1,3,3,3-tetrafluoropropene.

The invention also relates to the use of the abovementioned ternary composition as a heat transfer fluid in a vapor compression circuit.

The invention also relates to a heat transfer composition comprising the abovementioned ternary composition as a heat transfer fluid, and one or more additives chosen from lubricants, stabilizers, surfactants, tracer agents, fluorescers, odorants and solubilizers, and mixtures thereof.

The invention also relates to a heat transfer installation comprising a vapor compression circuit containing the above-mentioned ternary composition as a heat transfer fluid, or containing an abovementioned heat transfer composition.

According to one embodiment, this installation is chosen from mobile or stationary heat-pump heating, air-conditioning, refrigeration and freezing installations.

The invention also relates to a process for heating or cooling a fluid or a body using a vapor compression circuit containing a heat transfer fluid, said process successively comprising evaporation of the heat transfer fluid, compression of the heat transfer fluid, condensation of the heat fluid and depressurization of the heat transfer fluid, and the heat transfer fluid being the abovementioned ternary composition.

According to one embodiment of the heating or cooling process, this process is a process for cooling a fluid or a body, in which the temperature of the cooled fluid or body is from -40°C . to -10°C ., preferably from -35°C . to -25°C . and more particularly preferably from -30°C . to -20°C ., and in which the heat transfer fluid comprises:

from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene, preferably from 18% to 25% of difluoromethane, from 8% to 20% of pentafluoroethane and from 55% to 74% of 2,3,3,3-tetrafluoropropene; or from 15% to 50% of difluoromethane, from 5% to 20% of pentafluoroethane and from 30% to 80% of 1,3,3,3-tetrafluoropropene, preferably from 30% to 40% of difluoromethane, from 8% to 20% of pentafluoroethane and from 40% to 62% of 1,3,3,3-tetrafluoropropene.

According to another embodiment of the heating or cooling process, this process is a process for cooling a fluid or a body, in which the temperature of the cooled fluid or body is from -15°C . to 15°C ., preferably from -10°C . to 10°C . and more particularly preferably from -5°C . to 5°C ., and in which the heat transfer fluid comprises:

from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene, preferably from 18% to 25% of difluoromethane, from 8% to 20% of pentafluoroethane and from 55% to 74% of 2,3,3,3-tetrafluoropropene; or from 5% to 30% of difluoromethane, from 5% to 20% of pentafluoroethane and from 50% to 90% of 1,3,3,3-tetrafluoropropene, preferably from 5% to 20% of difluoromethane, from 5% to 20% of pentafluoroethane and from 60% to 90% of 1,3,3,3-tetrafluoropropene; or from 20% to 40% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 75% of 1,3,3,3-tetrafluoropropene, preferably from 25% to 40% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 70% of 1,3,3,3-tetrafluoropropene.

According to another embodiment of the heating or cooling process, this process is a process for heating a fluid or a body, in which the temperature of the heated fluid or body is from 30°C . to 80°C ., preferably from 35°C . to 55°C . and more particularly preferably from 40°C . to 50°C ., and in which the heat transfer fluid comprises:

from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene, preferably from 18% to 25% of difluoromethane, from 8% to 20% of pentafluoroethane and from 55% to 74% of 2,3,3,3-tetrafluoropropene; or from 5% to 30% of difluoromethane, from 5% to 20% of pentafluoroethane and from 50% to 90% of 1,3,3,3-tetrafluoropropene, preferably from 5% to 20% of difluoromethane, from 5% to 20% of pentafluoroethane and from 60% to 90% of 1,3,3,3-tetrafluoropropene; or

from 20% to 40% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 75% of 1,3,3,3-tetrafluoropropene, preferably from 25% to 40% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 70% of 1,3,3,3-tetrafluoropropene.

The invention also relates to a process for reducing the environmental impact of a heat transfer installation comprising a vapor compression circuit containing an initial heat transfer fluid, said process comprising a step of replacing the initial heat transfer fluid in the vapor compression circuit with a final transfer fluid, the final transfer fluid having a lower GWP than the initial heat transfer fluid, in which the final heat transfer fluid is the abovementioned ternary composition.

According to one embodiment of this process for reducing the environmental impact, the initial heat transfer fluid is a ternary mixture of 52% of 1,1,1-trifluoroethane, 44% of pentafluoroethane and 4% of 1,1,1,2-tetrafluoroethane or a ternary mixture of 52% of 1,1,1,2-tetrafluoroethane, 25% of pentafluoroethane and 23% of difluoromethane, and the final heat transfer fluid comprises:

from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene, preferably from 18% to 25% of difluoromethane, from 8% to 20% of pentafluoroethane and from 55% to 74% of 2,3,3,3-tetrafluoropropene; or from 15% to 50% of difluoromethane, from 5% to 20% of pentafluoroethane and from 30% to 80% of 1,3,3,3-tetrafluoropropene, preferably from 30% to 40% of difluoromethane, from 8% to 20% of pentafluoroethane and from 40% to 62% of 1,3,3,3-tetrafluoropropene; or from 20% to 40% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 75% of 1,3,3,3-tetrafluoropropene, preferably from 25% to 40% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 70% of 1,3,3,3-tetrafluoropropene.

According to another embodiment of this process for reducing the environmental impact, the initial heat transfer fluid is 1,1,1,2-tetrafluoroethane and the final heat transfer fluid comprises:

from 5% to 30% of difluoromethane, from 5% to 20% of pentafluoroethane and from 50% to 90% of 1,3,3,3-tetrafluoropropene, preferably from 5% to 20% of difluoromethane, from 5% to 20% of pentafluoroethane and from 60% to 90% of 1,3,3,3-tetrafluoropropene.

The present invention makes it possible to overcome the drawbacks of the prior art. It more particularly provides heat transfer fluids with a relatively low GWP, which have better energy performance qualities than the known heat transfer fluids.

This is accomplished by means of ternary mixtures comprising HFC-32, HFC-125 and tetrafluoropropene in the proportions indicated above.

According to certain particular embodiments, the invention also has one or preferably several of the advantageous features listed below.

The heat transfer fluids of the invention have a performance coefficient higher than the reference coolants R404a, R407c and/or HFC-134, in the same type of applications.

The capacity of the heat transfer fluids of the invention is greater than or equal to that of the reference coolants, in the same type of applications. Correlatively, the invention makes it possible to reduce the GWP of existing systems comprising one of the above reference coolants, and to do so while improving to a large extent the per-

5

formance qualities of these systems, by replacing the reference coolants with the heat transfer fluids of the invention.

The heat transfer fluids of the invention have a performance coefficient greater than that of the ternary mixtures of HFC-32, HFC-125 and HFO-1234ze or HFO-1234yf which are described in documents US 2009/0 250 650 and WO 2010/002 014.

Certain heat transfer fluids according to the invention make it possible to obtain a compressor outlet temperature which is lower than that obtained with the heat transfer fluids of the prior art.

According to the invention, the global warming potential (GWP) is defined relative to carbon dioxide and relative to a duration of 100 years, according to the method indicated in "The scientific assessment of ozone depletion, 2002, a report of the World Meteorological Association's Global Ozone Research and Monitoring Project".

DESCRIPTION OF EMBODIMENTS OF THE INVENTION

The invention is now described in greater detail and in a non-limiting manner in the description that follows.

The term "heat transfer compound" or, respectively, "heat transfer fluid" (or coolant fluid) means a compound or, respectively, a fluid which is capable of absorbing heat by evaporating at low temperature and low pressure and of expelling heat by condensing at high temperature and high pressure, in a vapor compression circuit. In general, a heat transfer fluid may comprise only one, two, three or more than three heat transfer compounds.

The term "heat transfer composition" means a composition comprising a heat transfer fluid and optionally one or more additives that are not heat transfer compounds for the envisioned application.

The heat transfer process according to the invention is based on the use of an installation comprising a vapor compression circuit which contains a heat transfer fluid. The heat transfer process may be a process for heating or cooling a fluid or a body.

The vapor compression circuit containing a heat transfer fluid comprises at least one evaporator, a compressor, a condenser and a pressure reducer, and also lines for transporting heat transfer fluid between these elements. The evaporator and the condenser comprise a heat exchanger that allows heat exchange between the heat transfer fluid and another fluid or body.

As compressor, use may be made especially of a centrifugal compressor containing one or more stages or a centrifugal minicompressor. Rotary, piston or screw compressors may also be used. The compressor may be driven by an electric motor or by a gas turbine (for example fed with a vehicle's exhaust gases, for mobile applications) or by gearing.

The installation may comprise a turbine for generating electricity (Rankine cycle).

The installation may also optionally comprise at least one heat transfer fluid circuit used for transmitting heat (with or without a change of state) between the heat transfer fluid circuit and the fluid or body to be heated or cooled.

The installation may also optionally comprise two (or more) vapor compression circuits, containing identical or different heat transfer fluids. For example, the vapor compression circuits may be coupled together.

The vapor compression circuit operates according to a standard vapor compression cycle. The cycle comprises change of state of the heat transfer fluid from a liquid phase

6

(or two-phase liquid/vapor) to a vapor phase at a relatively low pressure, and then compression of the fluid in the vapor phase to a relatively high pressure, change of state (condensation) of the heat transfer fluid from the vapor phase to the liquid phase at a relatively high pressure, and reduction of the pressure to recommence the cycle.

In the case of a cooling process, heat derived from the fluid or body that is cooled (directly or indirectly, via a heat transfer fluid) is absorbed by the heat transfer fluid, during its evaporation, at a relatively low temperature relative to the environment. The cooling processes comprise air-conditioning processes (with mobile installations, for example in vehicles, or stationary installations) and refrigeration and freezing or cryogenic processes.

In the case of a heating process, heat is transferred (directly or indirectly, via a heat transfer fluid) from the heat transfer fluid, during its condensation, to the fluid or body that is heated, at a relatively high temperature relative to the environment. The installation for performing the heat transfer is known in this case as a "heat pump".

It is possible to use any type of heat exchanger for the implementation of the heat transfer fluids according to the invention, and especially co-current heat exchangers.

However, according to one preferred embodiment, the invention envisages that the cooling and heating processes, and the corresponding installations, comprise a counter-current heat exchanger, either at the condenser or at the evaporator. Specifically, the heat transfer fluids according to the invention are particularly efficient with counter-current heat exchangers. Preferably, both the evaporator and the condenser comprise a counter-current heat exchanger.

According to the invention, the term "counter-current heat exchanger" means a heat exchanger in which heat is exchanged between a first fluid and a second fluid, the first fluid at the exchanger inlet exchanging heat with the second fluid at the exchanger outlet, and the first fluid at the exchanger outlet exchanging heat with the second fluid at the exchanger inlet.

For example, counter-current heat exchangers comprise devices in which the stream of the first fluid and the stream of the second fluid are in opposite or virtually opposite directions. Exchangers operating in cross-current mode with a counter-current tendency are also included among the counter-current heat exchangers for the purposes of the present application.

The meaning of the various abbreviations used to denote the various chemical compounds mentioned in the application is as follows:

HFC-134a: 1,1,1,2-tetrafluoroethane;

HFC-125: pentafluoroethane;

HFC-32: difluoromethane;

HFO-1234ze: 1,3,3,3-tetrafluoropropene;

HFO-1234yf: 2,3,3,3-tetrafluoropropene.

The heat transfer fluids used in the invention are the following ternary mixtures:

1) HFC-32, HFC-125 and HFO-1234ze; and

2) HFC-32, HFC-125 and HFO-1234yf.

The term "ternary mixture" means a composition consisting essentially of the three mentioned compounds, i.e. in which the three mentioned compounds represent at least 99% (preferably at least 99.5% or even at least 99.9%) of the composition.

Unless otherwise mentioned, throughout the application, the indicated proportions of compounds are given as mass percentages.

HFO-1234ze may be in cis or trans form (preferably trans) or may be a mixture of these two forms.

In the above compositions, HFC-32 is present in an amount of from 5% to 50%, HFC-125 is present in an amount of from 2% to 20% and HFO-1234yf or HFO-1234ze is present in an amount of from 30% to 90%.

For use in low-temperature refrigeration processes, i.e. those in which the temperature of the cooled fluid or body is from -40°C. to -10°C. , preferably from -35°C. to -25°C. and more particularly preferably from -30°C. to -20°C. (ideally about -25°C.), it has been found that the compositions that are the most efficient for replacing R404a are the following:

for composition 1): from 15% to 50% HFC-32, from 5% to 20% HFC-125 and from 30% to 80% HFO-1234ze, and preferably from 30% to 40% HFC-32, from 8% to 20% HFC-125 and from 40% to 62% HFO-1234ze;

for composition 2): from 15% to 35% HFC-32, from 5% to 20% HFC-125 and from 45% to 80% HFO-1234yf, and preferably from 18% to 25% HFC-32, from 8% to 20% HFC-125 and from 55% to 74% HFO-1234yf.

For a use in:

cooling processes at moderate temperature, i.e. those in which the temperature of the cooled fluid or body is from -15°C. to 15°C. , preferably from -10°C. to 10°C. and more particularly preferably from -5°C. to 5°C. (ideally about 0°C.), and also

heating processes at moderate temperature, i.e. those in which the temperature of the heated fluid or body is from 30°C. to 80°C. , preferably from 35°C. to 55°C. and more particularly preferably from 40°C. to 50°C. (ideally about 45°C.),

it has been found that the compositions that are the most efficient for replacing HFC-134a are the following:

for composition 1): from 5% to 30% HFC-32, from 5% to 20% HFC-125 and from 50% to 90% HFO-1234ze, and preferably from 5% to 20% HFC-32, from 5% to 20% HFC-125 and from 60% to 90% HFO-1234ze.

For a use in:

cooling processes at moderate temperature, i.e. those in which the temperature of the cooled fluid or body is from -15°C. to 15°C. , preferably from -10°C. to 10°C. and more particularly preferably from -5°C. to 5°C. (ideally about 0°C.), and also

heating processes at moderate temperature, i.e. those in which the temperature of the heated fluid or body is from 30°C. to 80°C. , preferably from 35°C. to 55°C. and more particularly preferably from 40°C. to 50°C. (ideally about 45°C.),

it has been found that the compositions that are the most efficient for replacing R404a or R407c are the following:

for composition 1): from 20% to 40% HFC-32, from 5% to 20% HFC-125 and from 40% to 75% HFO-1234ze, and preferably from 25% to 40% HFC-32, from 5% to 20% HFC-125 and from 40% to 70% HFO-1234ze;

for composition 2): from 15% to 35% HFC-32, from 5% to 20% HFC-125 and from 45% to 80% HFO-1234yf, and preferably from 18% to 25% HFC-32, from 8% to 20% HFC-125 and from 55% to 74% HFO-1234yf.

In the "low-temperature refrigeration" processes mentioned above, the inlet temperature of the heat transfer fluid at the evaporator is preferably from -45°C. to -15°C. , especially from -40°C. to -20°C. and more particularly preferably from -35°C. to -25°C. , for example about -30°C. ; and the temperature of the start of condensation of the heat transfer fluid at the condenser is preferably from 25°C. to 80°C. , especially from 30°C. to 60°C. and more particularly preferably from 35°C. to 55°C. , for example about 40°C.

In the "moderate-temperature cooling" processes mentioned above, the inlet temperature of the heat transfer fluid at the evaporator is preferably from -20°C. to 10°C. , especially from -15°C. to 5°C. and more particularly preferably from -10°C. to 0°C. , for example about -5°C. ; and the temperature of the start of condensation of the heat transfer fluid at the condenser is preferably from 25°C. to 80°C. , especially from 30°C. to 60°C. and more particularly preferably from 35°C. to 55°C. , for example about 50°C. These processes may be refrigeration or air-conditioning processes.

In the "moderate-temperature heating" processes mentioned above, the inlet temperature of the heat transfer fluid at the evaporator is preferably from -20°C. to 10°C. , especially from -15°C. to 5°C. and more particularly preferably from -10°C. to 0°C. , for example about -5°C. ; and the temperature of the start of condensation of the heat transfer fluid at the condenser is preferably from 25°C. to 80°C. , especially from 30°C. to 60°C. and more particularly preferably from 35°C. to 55°C. , for example about 50°C.

The heat transfer fluids mentioned above are not quasi-azeotropic and are highly efficient when they are correctly coupled with a counter-current heat exchanger (with an approximately constant difference in temperature with the second fluid in the exchanger).

Each heat transfer fluid above may be mixed with one or more additives to give the heat transfer composition effectively circulating in the vapor compression circuit. The additives may be chosen especially from lubricants, stabilizers, surfactants, tracer agents, fluorescers, odorants and solubilizers, and mixtures thereof.

The stabilizer(s), when they are present, preferably represent not more than 5% by mass in the heat transfer composition. Among the stabilizers, mention may be made especially of nitromethane, ascorbic acid, terephthalic acid, azoles such as tolutriazole or benzotriazole, phenolic compounds such as tocopherol, hydroquinone, t-butylhydroquinone, 2,6-di-tert-butyl-4-methylphenol, epoxides (optionally fluorinated or perfluorinated alkyl or alkenyl or aromatic) such as n-butyl glycidyl ether, hexanediol diglycidyl ether, allyl glycidyl ether, butylphenyl glycidyl ether, phosphites, phosphonates, thiols and lactones.

Lubricants that may especially be used include oils of mineral origin, silicone oils, paraffins, naphthenes, synthetic paraffins, alkylbenzenes, poly- α -olefins, polyalkylene glycols, polyol esters and/or polyvinyl ethers.

As tracer agents (capable of being detected), mention may be made of hydrofluorocarbons, deuterated hydrofluorocarbons, deuterated hydrocarbons, perfluorocarbons, fluoroethers, brominated compounds, iodinated compounds, alcohols, aldehydes, ketones and nitrous oxide, and combinations thereof. The tracer agent is different than the heat transfer compound(s) of which the heat transfer fluid is composed.

Solubilizers that may be mentioned include hydrocarbons, dimethyl ether, polyoxyalkylene ethers, amides, ketones, nitriles, chlorocarbons, esters, lactones, aryl ethers, fluoroethers and 1,1,1-trifluoroalkanes. The solubilizer is different than the heat transfer compound(s) of which the heat transfer fluid is composed.

Fluorescers that may be mentioned include naphthalimides, perylenes, coumarins, anthracenes, phenanthracenes, xanthenes, thioxanthenes, naphthoxanthenes and fluorescens, and derivatives and combinations thereof.

Odorants that may be mentioned include alkyl acrylates, allyl acrylates, acrylic acids, acryl esters, alkyl ethers, alkyl esters, alkynes, aldehydes, thiols, thioethers, disulfides, allyl-isothiocyanates, alkanolic acids, amines, norbornenes, nor-

bornene derivatives, cyclohexene, heterocyclic aromatic compounds, ascaridol and o-methoxy(methyl)phenol, and combinations thereof.

The compositions according to the invention may also be useful as expanders, aerosols or solvents.

EXAMPLES

The examples that follow illustrate the invention without limiting it.

Example 1

Method for Calculating the Properties of the Heat Transfer Fluids in the Various Envisioned Configurations

The RK-Soave equation is used to calculate the densities, enthalpies, entropies and liquid vapor equilibrium data of the mixtures. The use of this equation requires knowledge of the properties of the pure substances used in the mixtures in question and also the interaction coefficients for each binary.

The data required for each pure substance are the boiling point, the critical temperature and the critical pressure, the pressure curve as a function of the temperature from the boiling point up to the critical point, the saturated liquid density and the saturated vapor density as a function of the temperature.

The data for HFCs are published in ASHRAE Handbook 2005, chapter 20, and are also available under Refprop (software developed by NIST for calculating the properties of coolant fluids).

The data of the temperature-pressure curve for the HFOs are measured by the static method. The critical temperature and the critical pressure are measured with a C80 calorimeter sold by Setaram. The densities, at saturation as a function of the temperature, are measured by the vibrating-tube densimeter technique developed by the laboratories of the Ecole des Mines de Paris.

The RK-Soave equation uses binary interaction coefficients to represent the behavior of the products as mixtures. The coefficients are calculated as a function of the liquid vapor equilibrium experimental data.

The technique used for the liquid vapor equilibrium measurements is the analytical static cell method. The equilibrium cell comprises a sapphire tube and is equipped with two ROLSITM electromagnetic samplers. It is immersed in a cryothermostatic bath (HUBER HS40). A field-driven magnetic stirrer rotating at variable speed is used to accelerate the arrival at equilibrium. The analysis of the samples is performed by gas chromatography (HP5890 series II) using a katharometer (TCD).

The liquid vapor equilibrium measurements on the binary HFC-32/HFO-1234ze are performed for the following isotherm: 15° C.

The liquid vapor equilibrium measurements on the binary HFC-32/HFO-1234yf are performed for the following isotherms: 70° C., 30° C., -10° C.

The liquid vapor equilibrium data for the binary HFC-32/HFC-125 are available under Refprop. Three isotherms (-30° C., 0° C. and 30° C.) are used to calculate the interaction coefficients for this binary.

The liquid vapor equilibrium measurements on the binary HFC-125/HFO-1234yf are performed for the following isotherms: -15° C., 0° C.

The liquid vapor equilibrium measurements on the binary HFC-125/HFO-1234ze are performed for the following isotherms: 25° C., 0° C.

A compression system equipped with an evaporator and counter-current condenser, a screw compressor and a pressure reducer is considered.

The system operates with 15° C. of overheating and 5° C. of undercooling. The minimum temperature difference between the secondary fluid and the coolant fluid is considered of the order of 5° C.

The isentropic yield of the compressors is a function of the compression rate. This yield is calculated according to the following equation:

$$\eta_{isen} = a - b(\tau - c)^2 - \frac{d}{\tau - e} \quad (1)$$

For a screw compressor, the constants a, b, c, d and e of equation (1) of the isentropic yield are calculated according to the standard data published in the *Handbook of air conditioning and refrigeration*, page 11.52.

The coefficient of performance (COP) is defined as being the working power provided by the system to the power provided or consumed by the system.

The Lorenz coefficient of performance (COP_{Lorenz}) is a reference coefficient of performance. It is a function of temperatures and is used to compare the COPs of different fluids.

The Lorenz coefficient of performance is defined as follows (the temperatures T are in K):

$$T_{mean}^{condenser} = T_{inlet}^{condenser} - T_{outlet}^{condenser} \quad (2)$$

$$T_{mean}^{evaporator} = T_{outlet}^{evaporator} - T_{inlet}^{evaporator} \quad (3)$$

The Lorenz COP in the case of conditioned air and of refrigeration is:

$$COP_{Lorenz} = \frac{T_{mean}^{evaporator}}{T_{mean}^{condenser} - T_{mean}^{evaporator}} \quad (4)$$

The Lorenz COP in the case of heating is:

$$COP_{Lorenz} = \frac{T_{mean}^{condenser}}{T_{mean}^{condenser} - T_{mean}^{evaporator}} \quad (5)$$

For each composition, the coefficient of performance of the Lorenz cycle is calculated as a function of the corresponding temperatures.

In low-temperature refrigeration mode, the compression system operates between a coolant fluid inlet temperature at the evaporator of -30° C. and a coolant fluid inlet temperature at the condenser of 40° C. The system provides cold at -25° C.

In moderate-temperature heating mode, the compression system operates between a coolant fluid inlet temperature at the evaporator of -5° C. and a temperature of the start of condensation of the coolant fluid at the condenser of 50° C. The system provides heat at 45° C.

In moderate-temperature cooling mode, the compression system operates between a coolant fluid inlet temperature at the evaporator of -5° C. and a temperature of the start of condensation of the coolant fluid at the condenser of 50° C. The system provides cold at 0° C.

US 9,359,540 B2

11

In the tables that follow, “Evap. outlet temp.” denotes the temperature of the fluid at the evaporator outlet, “Comp. outlet temp.” denotes the temperature of the fluid at the compressor outlet, “Cond. outlet T” denotes the temperature of the fluid at the condenser outlet, “evap. P” denotes the pressure of the fluid in the evaporator, “cond. P” denotes the pressure of the fluid in the condenser, “Rate (p/p)” denotes the compression rate, “Glide” denotes the temperature glide, “comp. yield” denotes the compressor yield, “% CAP” denotes the volumetric capacity of the fluid relative to the

12

reference fluid indicated on the first line, and “%COP/COP_{Lorenz}” denotes the ratio of the COP of the system relative to the COP of the corresponding Lorenz cycle.

Example 2

Results for a Low-Temperature Refrigeration, Comparison with R404a

HFC-32/HFC-125/HFO-1234ze mixture:												
Composition (%)			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate p/p	Glide	Comp. yield	% CAP	% COP/COP-Lorenz
R404A			-30	100	40	2.1	18.1	8.8	0.46	53.8	100	32
HFC-32	HFC-125	HFO-1234ze										
30	50	20	-25	117	35	2.2	18.4	8.2	5.14	58.1	131	38
23	25	52	-23	112	32	1.6	13.4	8.4	7.36	56.9	103	40
35	8	57	-22	121	32	1.7	14.0	8.2	7.86	58.4	113	41
40	8	52	-22	125	32	1.8	14.9	8.1	7.70	59.2	121	42
35	12	53	-22	121	32	1.8	14.4	8.2	7.73	58.7	116	41
40	12	48	-22	124	32	1.9	15.3	8.1	7.53	59.4	124	42
30	16	54	-22	117	32	1.7	13.8	8.3	7.67	58.0	110	41
35	16	49	-22	120	32	1.8	14.7	8.1	7.58	59.0	118	41
40	16	44	-23	124	32	1.9	15.7	8.1	7.32	59.6	126	42
30	20	50	-22	117	32	1.7	14.2	8.2	7.56	58.4	113	41
35	20	45	-23	120	32	1.9	15.2	8.1	7.41	59.3	121	41
40	20	40	-23	124	33	2.0	16.1	8.1	7.08	59.6	129	41

In the preceding table, as in the following tables, the grayed lines correspond to the compositions disclosed in documents US 2009/0 250 650 or WO 2010/002 014 and the following lines correspond to the compositions according to the invention.

HFC-125/HFO-1234yf mixture:

Composition %			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate (p/p)	Glide	Comp. yield	% CAP	% COP/COP-Lorenz
R404A			-30	100	40	2.1	18.1	8.8	0.46	53.8	100	32
HFO-1234yf	HFC-32	HFC-125										
25	25	50	-28	114	37	2.3	19.7	8.6	2.39	55.7	125	35
60	15	25	-26	98	34	1.7	14.7	8.5	3.84	56.5	99	37
72	20	8	-25	100	33	1.7	14.4	8.3	4.63	58.1	102	39
70	22	8	-25	101	33	1.8	14.8	8.2	4.73	58.5	106	39
67	25	8	-25	103	33	1.9	15.4	8.1	4.79	59.0	111	39
62	30	8	-25	108	34	2.0	16.5	8.1	4.68	59.2	119	39
70	18	12	-26	99	33	1.7	14.3	8.3	4.37	57.4	100	38
68	20	12	-25	100	33	1.8	14.7	8.3	4.50	58.0	104	38
66	22	12	-25	102	33	1.8	15.1	8.2	4.58	58.3	107	39
63	25	12	-25	104	33	1.9	15.8	8.2	4.61	58.7	113	39
66	18	16	-26	99	33	1.8	14.6	8.3	4.27	57.4	101	38
64	20	16	-26	101	33	1.8	15.0	8.3	4.37	57.8	105	38
62	22	16	-26	102	33	1.9	15.5	8.2	4.43	58.2	109	38
59	25	16	-26	105	34	2.0	16.1	8.2	4.43	58.4	114	39
62	18	20	-26	100	33	1.8	15.0	8.4	4.16	57.3	103	38
60	20	20	-26	101	33	1.9	15.4	8.3	4.24	57.7	107	38
58	22	20	-26	103	34	1.9	15.8	8.3	4.27	58.0	110	38
55	25	20	-26	106	34	2.0	16.5	8.2	4.24	58.2	115	38

Results for a Moderate-Temperature Cooling,
Comparison with HFC-134a

HFC-32/HFC-125/HFO-1234ze mixture:

Composition (%)			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate (p/p)	Glide	Comp. yield	% CAP	% COP/ COP-Lorenz
R134a			-5	81	50	2.4	13.2	5.4	0.00	75.9	100	54
HFC-32	HFC-125	HFO-1234ze										
5	15	80	-1	74	44	2.5	12.2	4.8	4.02	78.5	106	56
5	18	77	-1	74	44	2.6	12.4	4.7	4.27	78.7	109	56
5	20	75	-1	74	44	2.7	12.6	4.7	4.42	78.8	111	56
10	5	85	0	76	43	2.7	12.5	4.6	5.10	79.1	116	58
10	15	75	1	77	43	2.9	13.3	4.5	5.52	79.5	124	57
10	18	72	1	77	43	3.0	13.6	4.5	5.64	79.6	127	57
10	20	70	1	76	43	3.1	13.7	4.5	5.72	79.6	128	57
15	5	80	1	79	42	3.1	13.5	4.4	6.32	79.8	132	58
15	15	70	1	79	42	3.3	14.4	4.3	6.43	80.0	139	58
15	18	67	1	79	42	3.4	14.7	4.3	6.47	80.1	142	58
15	20	65	2	79	42	3.5	14.9	4.3	6.50	80.1	144	58
20	5	75	2	81	42	3.4	14.6	4.3	7.01	80.2	146	59
20	15	65	2	81	42	3.7	15.5	4.2	6.95	80.4	154	58

Example 4

30

Results for a Moderate-Temperature Heating,
Comparison with HFC-134a

HFC-32/HFC-125/HFO-1234ze mixture:

Composition (%)			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate (p/p)	Glide	Comp. yield	% CAP	% COP/ COP-Lorenz
R134a			-5	81	50	2.4	13.2	5.4	0.00	75.9	100	63
HFC-32	HFC-125	HFO-1234ze										
5	15	80	-1	74	44	2.5	12.2	4.8	4.02	78.5	103	65
5	18	77	-1	74	44	2.6	12.4	4.7	4.27	78.7	106	65
5	20	75	-1	74	44	2.7	12.6	4.7	4.42	78.8	108	65
10	5	85	0	76	43	2.7	12.5	4.6	5.10	79.1	110	66
10	15	75	1	77	43	2.9	13.3	4.5	5.52	79.5	118	66
10	18	72	1	77	43	3.0	13.6	4.5	5.64	79.6	121	66
10	20	70	1	76	43	3.1	13.7	4.5	5.72	79.6	123	66
15	5	80	1	79	42	3.1	13.5	4.4	6.32	79.8	124	66
15	15	70	1	79	42	3.3	14.4	4.3	6.43	80.0	132	66
15	18	67	1	79	42	3.4	14.7	4.3	6.47	80.1	135	66
15	20	65	2	79	42	3.5	14.9	4.3	6.50	80.1	136	66
20	5	75	2	81	42	3.4	14.6	4.3	7.01	80.2	137	66
20	15	65	2	81	42	3.7	15.5	4.2	6.95	80.4	145	66

Results for a Moderate-Temperature Cooling,
Comparison with R404a and R407c

5

HFC-32/HFC-125/HFO-1234ze mixture:

Composition %			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate (p/p)	Glide	Comp. yield	% CAP	% COP/COP-Lorenz
R404A			-5	77	50	5.2	23.0	4.5	0.37	79.7	100	48
R407C			-1	89	45	4.5	19.8	4.4	4.46	79.9	114	56
HFC-32	HFC-125	HFO-1234ze										
30	50	20	0	89	45	5.7	23.6	4.2	4.88	80.5	133	54
23	25	52	2	83	42	4.2	17.3	4.1	7.04	80.6	109	58
25	5	70	2	84	42	3.8	15.7	4.2	7.38	80.5	102	59
25	15	60	2	84	42	4.0	16.7	4.1	7.23	80.6	107	59
30	5	65	3	87	42	4.1	16.8	4.1	7.54	80.7	110	59
40	5	55	2	92	42	4.7	18.9	4.0	7.45	80.9	124	59
40	15	45	2	92	43	5.0	20.1	4.0	7.04	80.9	129	58
40	18	42	2	92	43	5.1	20.5	4.0	6.88	80.9	131	58
40	20	40	2	92	43	5.2	20.8	4.0	6.77	80.9	132	58

Example 6

Results for a Moderate-Temperature Cooling,
Comparison with R404a

30

HFC-32/HFC-125/HFO-1234yf mixture:

Composition %			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate (p/p)	Glide	Comp. yield	% CAP	% COP/COP-Lorenz
R404A			-5	77	50	5.2	23.0	4.5	0.37	79.7	100	48
HFO-1234yf	HFC-32	HFC-125										
52	23	25	-1	82	45	5.1	21.1	4.1	4.32	80.6	117	54
60	15	25	-1	76	44	4.5	18.8	4.2	4.23	80.5	104	54
74	18	8	0	77	43	4.4	17.9	4.1	4.97	80.7	104	56
72	20	8	0	78	43	4.5	18.5	4.1	5.11	80.8	107	56
70	22	8	0	79	43	4.7	19.0	4.1	5.18	80.8	111	56
67	25	8	0	81	43	4.9	19.8	4.0	5.18	80.8	115	56
62	30	8	0	84	44	5.2	21.1	4.1	4.97	80.8	122	55
70	18	12	0	77	43	4.5	18.3	4.1	4.84	80.7	105	55
68	20	12	0	78	43	4.6	18.8	4.1	4.95	80.7	108	55
66	22	12	0	80	43	4.8	19.4	4.1	4.99	80.8	112	55
63	25	12	0	81	44	5.0	20.2	4.1	4.97	80.8	116	55
58	30	12	0	85	44	5.3	21.5	4.1	4.72	80.7	123	55
69	15	16	-1	76	43	4.3	17.9	4.2	4.47	80.6	100	55
66	18	16	0	77	43	4.5	18.7	4.1	4.70	80.7	106	55
64	20	16	0	79	44	4.7	19.3	4.1	4.78	80.7	109	55
62	22	16	0	80	44	4.8	19.8	4.1	4.80	80.7	113	55
59	25	16	0	82	44	5.0	20.6	4.1	4.75	80.7	117	55
65	15	20	-1	76	44	4.4	18.3	4.2	4.37	80.5	102	55
60	20	20	0	79	44	4.8	19.7	4.1	4.61	80.6	111	55
58	22	20	0	80	44	4.9	20.3	4.1	4.61	80.7	114	55

Results for a Moderate-Temperature Heating,
Comparison with R404a

HFC-32/HFC-125/HFO-1234yf mixture:

Composition %			Evap. outlet temp. (° C.)	Comp. outlet temp. (° C.)	Cond. outlet T (° C.)	Evap. P (bar)	Cond. P (bar)	Rate (p/p)	Glide	Comp. yield	% CAP	% COP/COP- Lorenz
R404A			-5	77	50	5.2	23.0	4.5	0.37	79.7	100	48
HFO- 1234yf	HFC- 32	HFC- 125										
32	23	25	-1	82	45	5.1	21.1	4.1	4.32	80.6	110	63
60	15	25	-1	76	44	4.5	18.8	4.2	4.23	80.5	98	63
72	20	8	0	78	43	4.5	18.5	4.1	5.11	80.8	100	64
70	22	8	0	79	43	4.7	19.0	4.1	5.18	80.8	103	64
67	25	8	0	81	43	4.9	19.8	4.0	5.18	80.8	108	64
68	20	12	0	78	43	4.6	18.8	4.1	4.95	80.7	102	64
66	22	12	0	80	43	4.8	19.4	4.1	4.99	80.8	105	64
66	18	16	0	77	43	4.5	18.7	4.1	4.70	80.7	100	64

The invention claimed is:

1. A ternary composition comprising, in mass percent:
from 5% to 35% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 30% to 90% of tetrafluoropropene.
2. The composition as claimed in claim 1, in which the tetrafluoropropene is 1,3,3,3-tetrafluoropropene.
3. The composition as claimed in claim 1, in which the tetrafluoropropene is 2,3,3,3-tetrafluoropropene.
4. The composition as claimed in claim 1, comprising, in mass percent:
from 15% to 35% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 45% to 80% of 2,3,3,3-tetrafluoropropene.
5. The composition as claimed in claim 1, comprising, in mass percent:
from 15% to 35% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 30% to 80% of 1,3,3,3-tetrafluoropropene.
6. The composition as claimed in claim 1, comprising, in mass percent:
from 5% to 30% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 50% to 90% of 1,3,3,3-tetrafluoropropene.
7. The composition as claimed in claim 1, comprising, in mass percent:
from 20% to 35% of difluoromethane;
from 5% to 20% of pentafluoroethane; and
from 40% to 75% of 1,3,3,3-tetrafluoropropene.
8. A vapor compression circuit comprising a composition as claimed in claim 1.
9. A heat transfer composition comprising the composition as claimed in claim 1 as a heat transfer fluid, further comprising one or more additives chosen from the group consisting of lubricants, stabilizers, surfactants, tracer agents, fluoresters, odorants and solubilizers, and mixtures thereof.
10. A heat transfer installation comprising a vapor compression circuit containing a composition as claimed in claim 1.
11. The installation as claimed in claim 10, chosen from mobile or stationary heat-pump heating, air-conditioning, refrigeration and freezing installations.
12. A process for heating or cooling a fluid or a body using a vapor compression circuit containing a heat transfer fluid,

said process successively comprising evaporation of the heat transfer fluid, compression of the heat transfer fluid, condensation of the heat fluid and depressurization of the heat transfer fluid, in which the heat transfer fluid is a composition as claimed in claim 1.

13. The process as claimed in claim 12, which is a process for cooling a fluid or a body, in which the temperature of the cooled fluid or body is from -40° C. to -10° C., and in which the heat transfer fluid comprises, in mass percent:
from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene; or
from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 30% to 80% of 1,3,3,3-tetrafluoropropene.
14. The process as claimed in claim 12, which is a process for cooling a fluid or a body, in which the temperature of the cooled fluid or body is from -15° C. to 15° C., and in which the heat transfer fluid comprises, in mass percent:
from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene; or
from 5% to 30% of difluoromethane, from 5% to 20% of pentafluoroethane and from 50% to 90% of 1,3,3,3-tetrafluoropropene; or
from 20% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 75% of 1,3,3,3-tetrafluoropropene.
15. The process as claimed in claim 12, which is a process for heating a fluid or a body, in which the temperature of the heated fluid or body is from 30° C. to 80° C., and in which the heat transfer fluid comprises, in mass percent:
from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene; or
from 5% to 30% of difluoromethane, from 5% to 20% of pentafluoroethane and from 50% to 90% of 1,3,3,3-tetrafluoropropene; or
from 20% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 75% of 1,3,3,3-tetrafluoropropene.
16. A process for reducing the environmental impact of a heat transfer installation comprising a vapor compression circuit containing an initial heat transfer fluid, said process comprising a step of replacing the initial heat transfer fluid in the vapor compression circuit with a final transfer fluid, the final transfer fluid having a lower GWP than the initial heat transfer fluid, in which the final heat transfer fluid is a composition as claimed in claim 1.

17. The process as claimed in claim 16, in which the initial heat transfer fluid is a ternary mixture of 52% of 1,1,1-trifluoroethane, 44% of pentafluoroethane and 4% of 1,1,1,2-tetrafluoroethane or a ternary mixture of 52% of 1,1,1,2-tetrafluoroethane, 25% of pentafluoroethane and 23% of difluoromethane, and in which the final heat transfer fluid comprises, in mass percent:

from 15% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 45% to 80% of 2,3,3,3-tetrafluoropropene; or

from 15% to 50% of difluoromethane, from 5% to 20% of pentafluoroethane and from 30% to 80% of 1,3,3,3-tetrafluoropropene; or

from 20% to 35% of difluoromethane, from 5% to 20% of pentafluoroethane and from 40% to 75% of 1,3,3,3-tetrafluoropropene.

18. The process as claimed in claim 16, in which the initial heat transfer fluid is 1,1,1,2-tetrafluoroethane, and in which the final heat transfer fluid comprises, in mass percent:

from 5% to 30% of difluoromethane, from 5% to 20% of pentafluoroethane and from 50% to 90% of 1,3,3,3-tetrafluoropropene.

19. A ternary composition comprising, in mass percent:

from 5% to 50% of difluoromethane;

from 8% to 20% of pentafluoroethane; and

from 55% to 74% of 2,3,3,3-tetrafluoropropene.

* * * * *